

ES128: Homework 2
Due in class on Wednesday, 03 March 2010

Problem 1

Show that the weak form of

$$\begin{aligned} \frac{d}{dx} \left(AE \frac{du}{dx} \right) + 2x &= 0 && \text{on } 1 < x < 3, \\ \sigma(1) = \left(E \frac{du}{dx} \right)_{x=1} &= 0.1, \\ u(3) &= 0.001 \end{aligned}$$

is given by

$$\int_1^3 \frac{dw}{dx} AE \frac{du}{dx} dx = -0.1(wA)_{x=1} + \int_1^3 2xw dx \quad \forall w \text{ with } w(3) = 0.$$

Problem 2

Consider the (steel) bar in Figure 1. The bar has a uniform thickness $t=1$ cm, Young's modulus $E=200 \times 10^9$ Pa, and weight density $\rho = 7 \times 10^3$ kg/m³. In addition to its self-weight, the plate is subjected to a point load $P=100$ N at its midpoint.

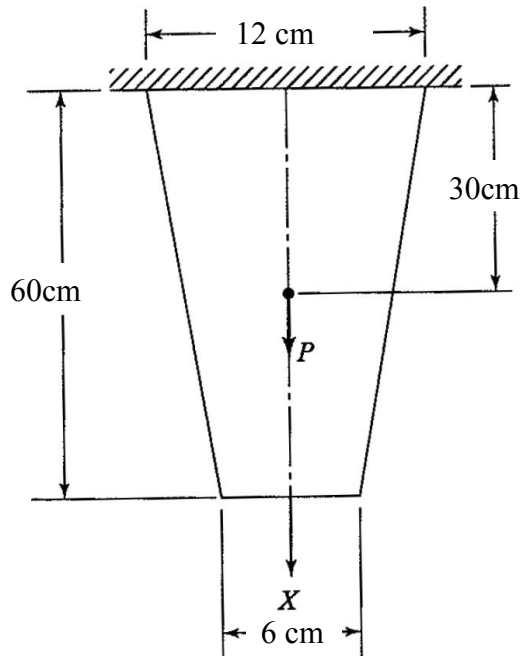


Figure 1

- Model the bar with two finite elements.
- Write down expressions for the element stiffness matrices and element body force vectors.
- Assemble the structural stiffness matrix \mathbf{K} and global load vector \mathbf{F} .
- Solve for the global displacement vector \mathbf{d} .
- Evaluate the stresses in each element.
- Determine the reaction force at the support.

Problem 3

Consider the mesh shown in Figure 2. The model consists of two linear displacement constant strain elements. The cross-sectional area is $A=1$, Young's modulus is E ; both are constant. A body force $b(x)=cx$ is applied.

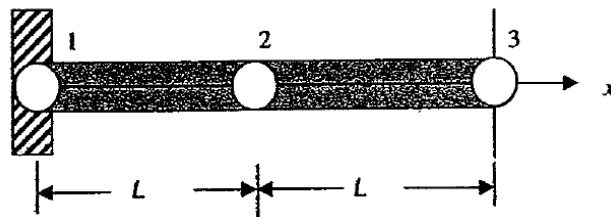


Figure 2

- Solve and plot $u(x)$ and $\varepsilon(x)$ for the FEM solution.
- Compare (by plotting) the finite element solution against the exact solution for the equation

$$E \frac{d^2 u}{dx^2} = -b(x) = -cx.$$

- Solve the above problem using a single quadratic displacement element.
- Compare the accuracy of stress and displacement at the right end with that of two linear displacement elements.
- Check whether the equilibrium equation and traction boundary condition are satisfied for the two meshes.